

ENERGY-BASED APPROACH TO LIMIT STATE CRITERIA OF CELLULAR MATERIALS

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ABSTRACT

Formulation of limit state criteria derived on micromechanical analysis for open-cell anisotropic media modeled as periodic beam structure is presented. Linear response and material strength from macroscopic perspective is described by energy based constitutive model.

PROBLEM DEFINITION

- multiscale modeling
- formulation for equivalent continuum

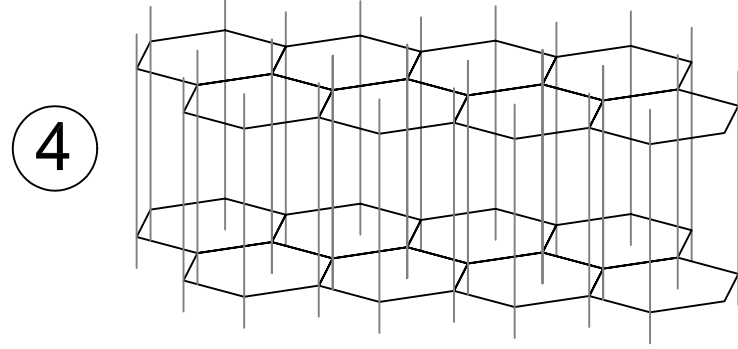
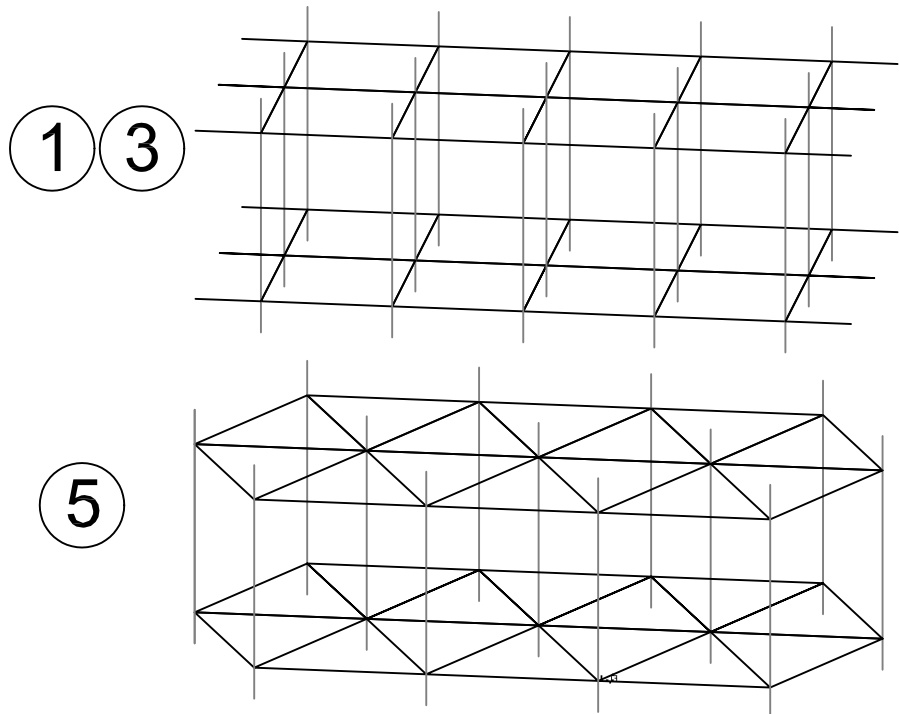
Rychlewski criterion

$$\frac{\Phi(\boldsymbol{\sigma}_I)}{\Phi_I^{cr}} + \frac{\Phi(\boldsymbol{\sigma}_{II})}{\Phi_{II}^{cr}} + \dots + \frac{\Phi(\boldsymbol{\sigma}_\rho)}{\Phi_\rho^{cr}} \leq 1 \quad \rho \leq 6$$

$\boldsymbol{\sigma}_I, \dots, \boldsymbol{\sigma}_\rho$ eigenstates

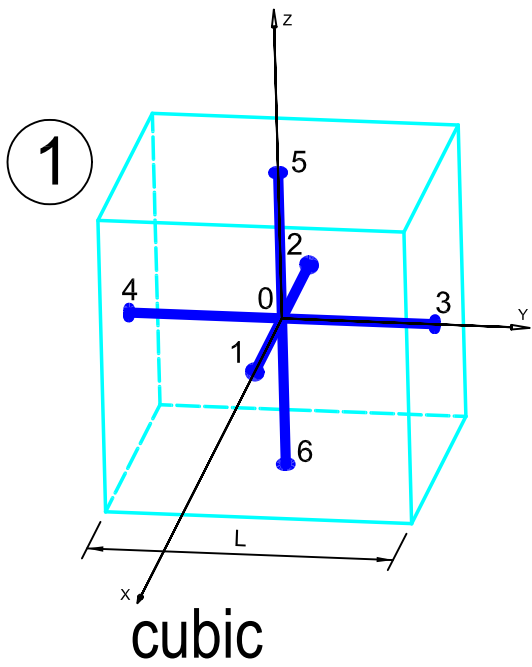
$\Phi_I^{cr}, \dots, \Phi_\rho^{cr}$ critical energy densities

Microstructures

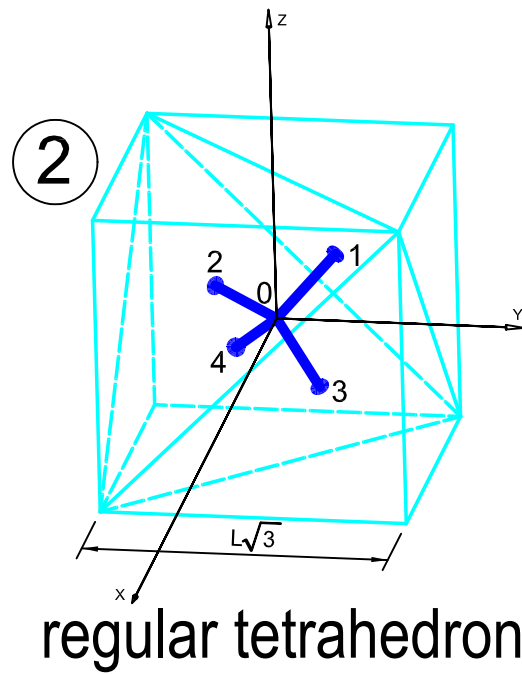


② 3-D foam

Representative unit cells

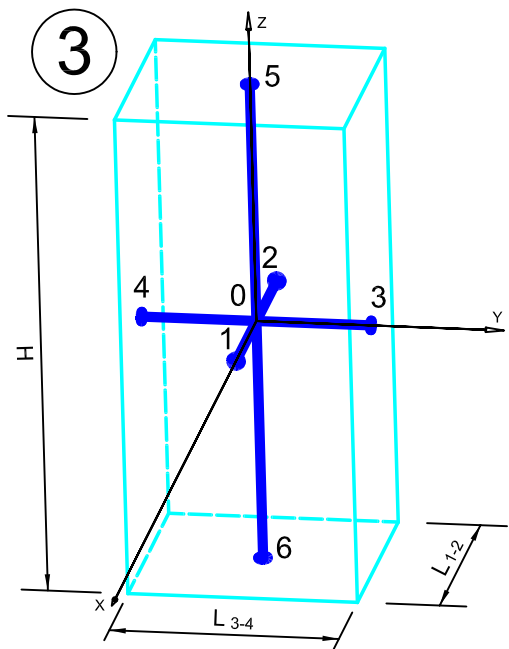


cubic



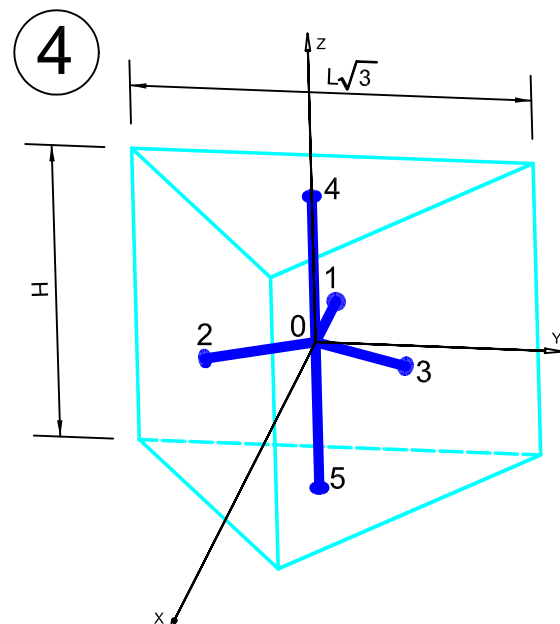
regular tetrahedron

cubic symmetry



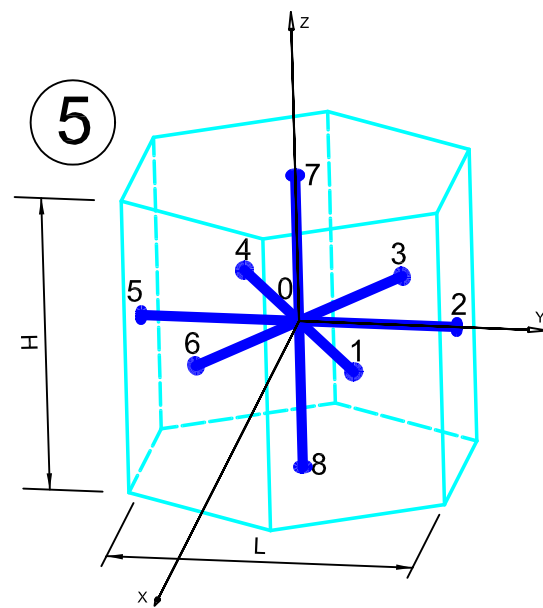
rectangular prism

orthotropic symmetry



trigonal prism

transversally isotropic symmetry



hexagonal prism

METHOD of structural mechanics

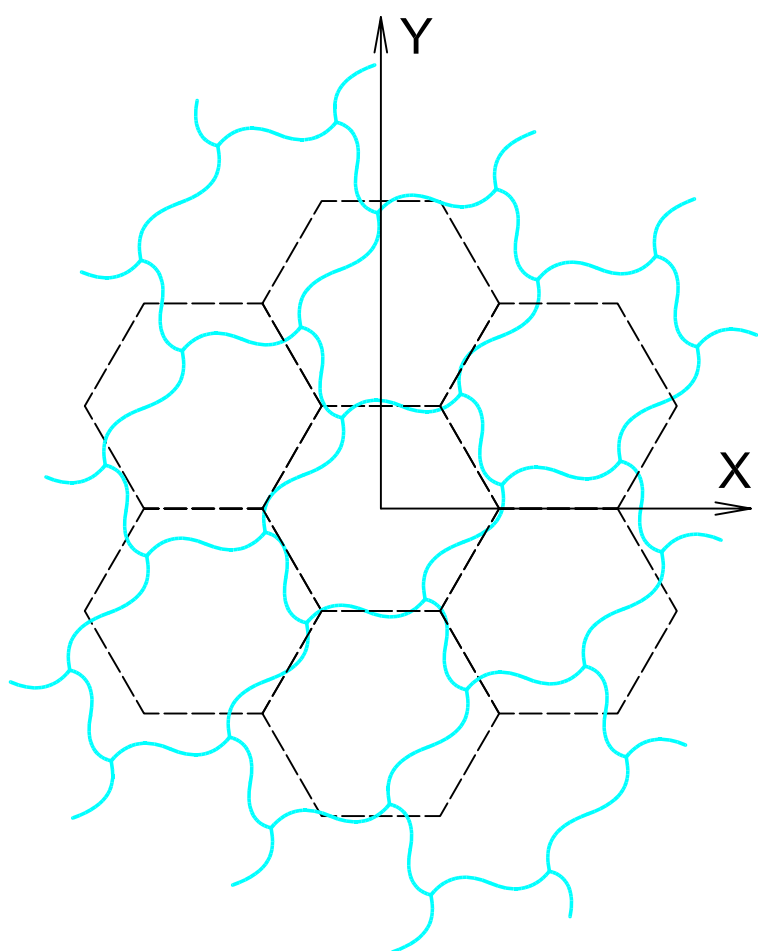
Framework of micromechanical analysis

- representative unit cell - kinematics
- mechanical model Timoshenko beam
- stress tensor definition
- stiffness tensor: Kelvin moduli and eigenstates
- micro-macro transition

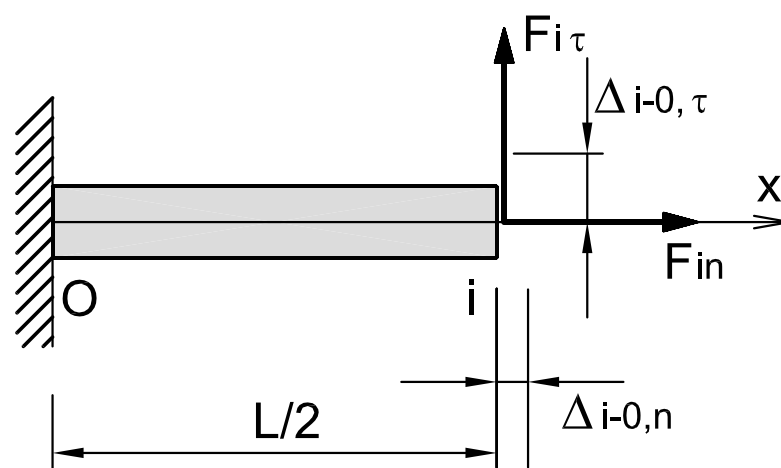
Kinematics

- affinity of nodes displacements
- uniform states of strains (macro-scale)

$$\Delta_i = \Delta_{i-0} + \boldsymbol{\psi} \times \mathbf{b}_i^0 + \Delta_0$$



Timoshenko beam model



axial strut compliance

$$c_n = s_n^{-1} = \frac{L}{2E_s A}$$

bending strut compliance

$$c_\tau = s_\tau^{-1} = \frac{L^3}{24E_s J} + \frac{L}{2G_s A_\tau}$$

s skeleton material

Displacement-force relations

$$F_{in} = \Delta_{i-0,n} \cdot S_n$$

$$F_{i\tau} = \Delta_{i-0,\tau} \cdot S_\tau$$

Stress definition for equivalent continuum

$$\boldsymbol{\sigma} = \frac{1}{V} \int_{V^s} \boldsymbol{\sigma}^s dV$$

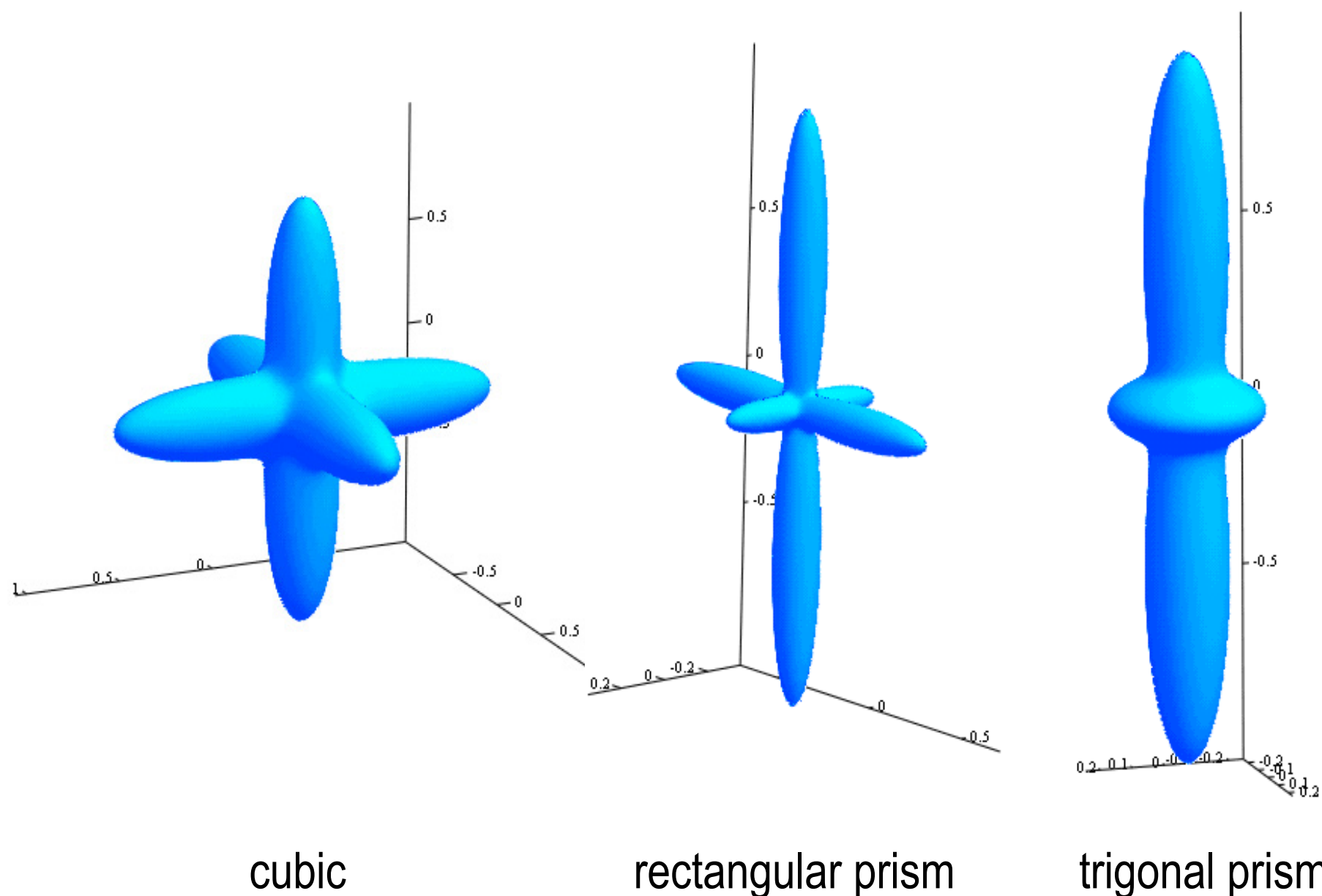
Hooke's law $\boldsymbol{\sigma} = \mathbf{S} \boldsymbol{\varepsilon}$,

\mathbf{S} stiffness tensor, λ_i eigenvalues

$\boldsymbol{\sigma}_i$ eigenstates $i = 1, \dots, \rho, \rho \leq 6$

Young's moduli $E(\mathbf{n}) = (\mathbf{n} \otimes \mathbf{n}) \cdot \mathbf{S} \cdot (\mathbf{n} \otimes \mathbf{n})$

$$E_r(\mathbf{n}) = \frac{E(\mathbf{n})}{E_{\max}}$$



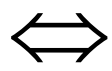
LIMIT STATES

skeleton

the limit states in the skeleton are calculated with the use of Huber criterion

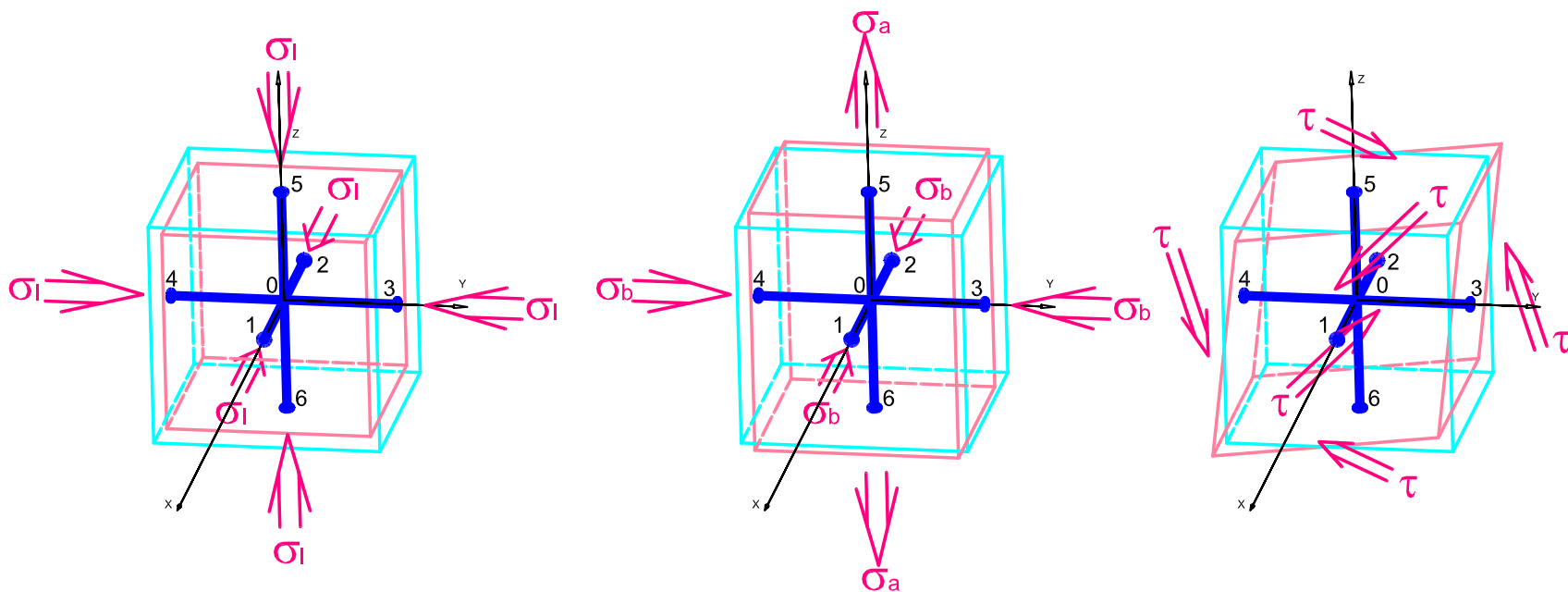
cellular material (as a continuum)

limit elastic eigenstates



$$\boldsymbol{\sigma}_{i,cr}^s \Leftrightarrow \boldsymbol{\sigma}_{i,cr}$$

↓ critical energy densities $\phi_{i,cr}$



$$\boldsymbol{\sigma}_I = \begin{bmatrix} \sigma_I & 0 & 0 \\ 0 & \sigma_I & 0 \\ 0 & 0 & \sigma_I \end{bmatrix}$$

$$\boldsymbol{\sigma}_{II} = \begin{bmatrix} \sigma_a & 0 & 0 \\ 0 & \sigma_b & 0 \\ 0 & 0 & \sigma_b \end{bmatrix}$$

$$\boldsymbol{\sigma}_{III} = \begin{bmatrix} 0 & \tau & \tau \\ \tau & 0 & \tau \\ \tau & \tau & 0 \end{bmatrix}$$

$$\sigma_s = R_e \Rightarrow \boldsymbol{\sigma}_i = \boldsymbol{\sigma}_{i,cr}$$

$$\phi_{i,cr} = \frac{1}{\lambda_i} \boldsymbol{\sigma}_{i,cr} \cdot \boldsymbol{\sigma}_{i,cr}$$

RESULTS

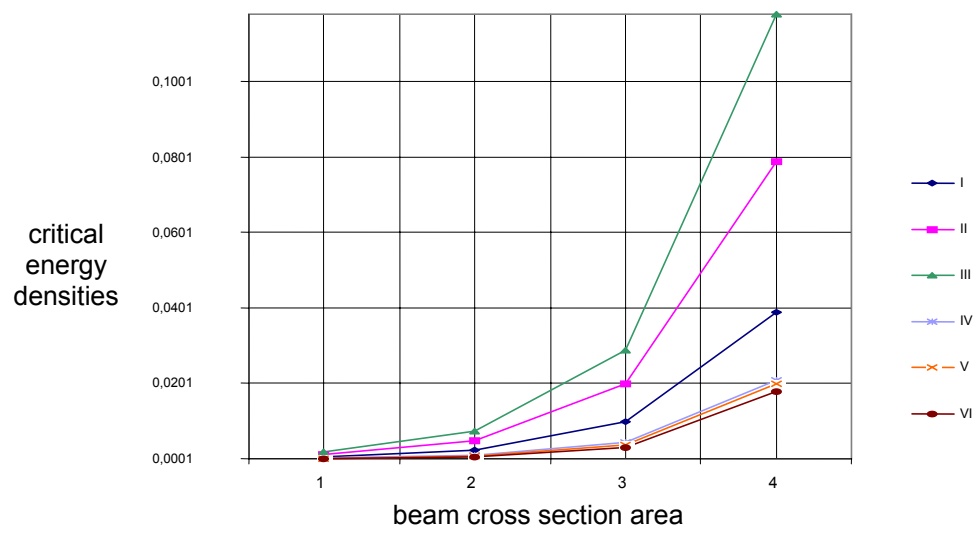
eigenvalues $\lambda_i = \lambda_i(s_n, s_\tau, H, L, A, \dots)$

cubic cell	$\lambda_I = \lambda_1 = \frac{s_n}{2L}, \quad \lambda_{II} = \lambda_2 = \lambda_3 = \frac{s_n}{2L}, \quad \lambda_{III} = \lambda_4 = \lambda_5 = \lambda_6 = \frac{s_\tau}{2L}$
rectangular prism	$\lambda_I = \lambda_1 = \frac{L_{1-2} s_{n1-2}}{2 L_{3-4} H}, \quad \lambda_{II} = \lambda_2 = \frac{L_{3-4} s_{n3-4}}{2 L_{1-2} H}, \quad \lambda_{III} = \lambda_3 = \frac{H s_{n5-6}}{2 L_{1-2} L_{3-4}}$ $\lambda_{IV} = \lambda_4 = \frac{\frac{2 H^2 s_{\tau 5-6}}{L_{3-4}^2 s_{\tau 3-4} + H^2 s_{\tau 5-6}} \frac{L_{3-4} s_{\tau 3-4}}{2}}{L_{1-2} H},$ $\lambda_V = \lambda_5 = \frac{\frac{2 H^2 s_{\tau 5-6}}{L_{1-2}^2 s_{\tau 1-2} + H^2 s_{\tau 5-6}} \frac{L_{1-2} s_{\tau 1-2}}{2}}{L_{3-4} H}$ $\lambda_{VI} = \lambda_6 = \frac{\frac{2 L_{3-4}^2 s_{\tau 3-4}}{L_{3-4}^2 s_{\tau 3-4} + L_{1-2}^2 s_{\tau 1-2}} \frac{L_{12} s_{\tau 1-2}}{2}}{L_{3-4} H}$
trigonal prism	$\lambda_I = \lambda_1 = \frac{2\sqrt{3} s_{nL}}{9L}, \quad \lambda_{II} = \lambda_3 = \frac{\sqrt{3} s_{nL}}{6H}, \quad \lambda_{III} = \lambda_2 = \lambda_6 = \frac{\sqrt{3} s_{nL} s_{\tau L}}{3H(s_{nL} + s_{\tau L})},$ $\lambda_{IV} = \lambda_4 = \lambda_5 = \frac{4\sqrt{3} H s_{\tau H} s_{\tau L}}{3L^2 s_{\tau L} + 4H^2 s_{\tau H}}$
foam cell	$\lambda_I = \lambda_1 = \frac{2(s_n + 2s_\tau)}{9\sqrt{3}L}, \quad \lambda_{II} = \lambda_2 = \lambda_3 = \frac{2(s_n - s_\tau)}{9\sqrt{3}L}, \quad \lambda_{III} = \lambda_{II}$

energy densities $\Phi_i = \Phi_i(R_e, s_n, s_\tau, H, L, A, \dots)$

cubic cell	$\Phi_I^{gr} = \frac{1}{\lambda_I} 3 \left(\frac{A R_e}{L^2} \right)^2, \quad \Phi_{II}^{gr} = \frac{1}{\lambda_{II}} \frac{2}{3} \left(\frac{A R_e}{L^2} \right)^2, \quad \Phi_{III}^{gr} = \frac{1}{\lambda_{III}} 24 \frac{I^2 R_e^2}{h^2 L^6}$
ortotropic cell	$\Phi_I^{gr} = \frac{1}{\lambda_I} \left(\frac{A R_e}{L_{3-4} H} \right)^2, \quad \Phi_{II}^{gr} = \frac{1}{\lambda_{II}} \left(\frac{A R_e}{L_{3-4} H} \right)^2, \quad \Phi_{III}^{gr} = \frac{1}{\lambda_{III}} \left(\frac{A R_e}{L_{1-2} L_{3-4}} \right)^2$ $\Phi_{IV}^{gr} = \frac{1}{\lambda_{IV}} 8 \frac{I^2 R_e^2}{h^2 L_{1-2}^2 L_{3-4}^2 H^2}, \quad \Phi_V^{gr} = \frac{1}{\lambda_V} 8 \frac{I^2 R_e^2}{h^2 L_{1-2}^2 L_{3-4}^2 H^2}$ $\Phi_{VI}^{gr} = \frac{1}{\lambda_{VI}} 8 \frac{I^2 R_e^2}{h^2 L_{1-2}^2 L_{3-4}^2 H^2}.$
trigonal prism	$\Phi_I^{gr} = \frac{1}{\lambda_I} \frac{(A R_e)^2}{2L H^3}, \quad \Phi_{II}^{gr} = \frac{1}{\lambda_{II}} \frac{64 H^2 A^2 R_e^2}{3 L^4}$ $\Phi_{III}^{gr} = \frac{26 \sqrt{3} s_{\tau L} (s_{nL} + s_{\tau L}) I^2 A^2 R_e^2}{9 s_{nL} H L^2 (2 I s_{nL} + 2 I s_{\tau L} + L s_{\tau L} h A)^2},$ $\Phi_{IV}^{gr} = \frac{16 \sqrt{3} (3 L^2 s_{\tau L} + 4 H^2 s_{\tau H}) I^2 R_e^2}{27 H^3 s_{\tau H} s_{\tau L} L^4 h^2}$
foam cell	$\Phi_I^{gr} = \frac{2}{3K} \left(\frac{\rho}{\rho^s} \right)^2 R_e^2, \quad \Phi_{II}^{gr} = \frac{R_e^2}{4G} f(A, L)$

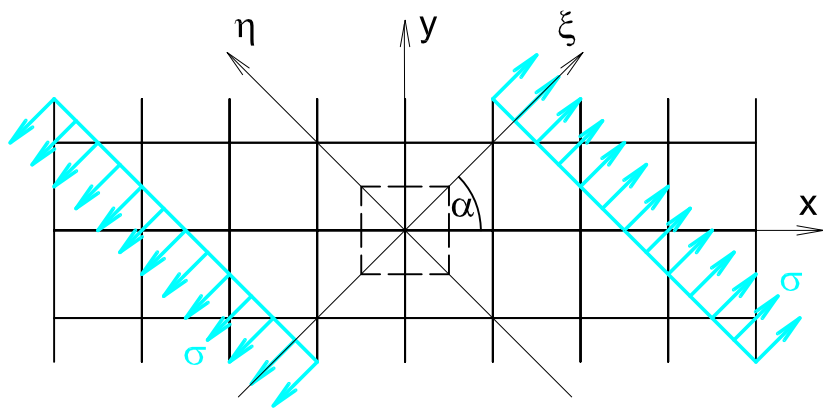
DISTRIBUTION OF ENERGY LIMITS



- dependence on microstructure parameters
- modelling possibilities

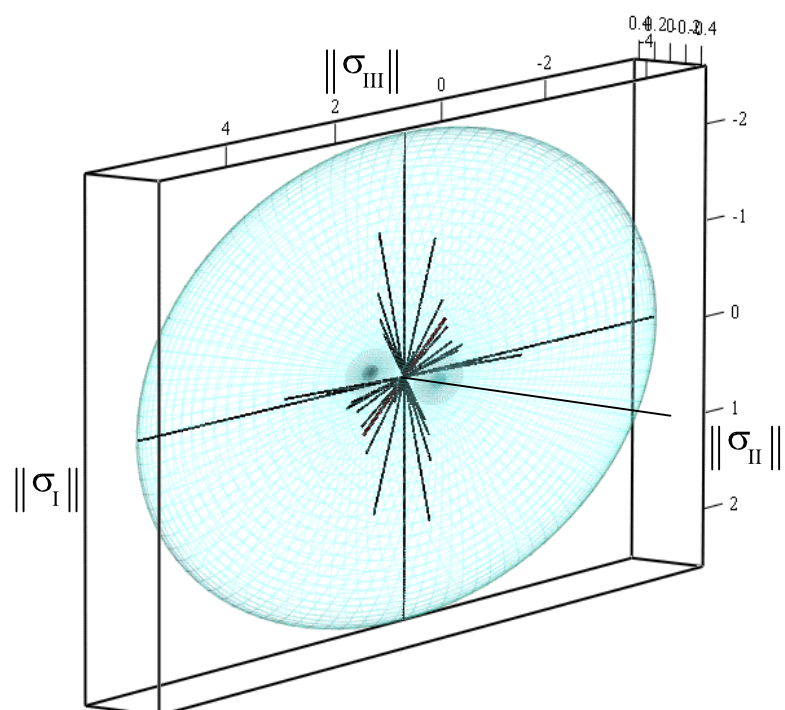
EXPERIMENTAL VERIFICATION OF ENERGY CRITERION

tension-compression tests in xy plane



$$\boldsymbol{\sigma}(\xi, \eta) = \begin{bmatrix} \sigma & 0 \\ 0 & 0 \end{bmatrix}$$

α - cell orientation angle with respect to tension direction



$$\boldsymbol{\sigma} = \boldsymbol{\sigma}_I + \boldsymbol{\sigma}_{II} + \boldsymbol{\sigma}_{III}$$

eigenstate decomposition of the plane stress state

theoretical prediction of σ_{cr}

$$\sigma_{cr} = \left[\frac{1}{3\lambda_I \Phi_{I,cr}} + \frac{4 - 3(\sin 2\alpha)^2}{6\lambda_{II} \Phi_{II,cr}} + \frac{(\sin 2\alpha)^2}{2\lambda_{III} \Phi_{III,cr}} \right]^{-1/2} \text{ for cubic cell}$$

CONCLUSION

- effective model of elastic behaviour and limit state of open-cell microstructures is proposed
- such an approach can be used for each type of (micro)structure (topology and morphology)
- the presented analysis can be extended for:
 - nonlinear elasticity
 - plastic analysis of struts (plastic hinges)
 - failure analysis
 - different models (plate model)
- theoretical background for experiment is given

Literature

- [1] **L.J. Gibson, M.F. Ashby** (1997). *Cellular Solids*, 2nd edition Cambridge University Press.
- [2] **J.Rychlewski** (1984). Unconventional approach to linear elasticity, *Arch. Mech.*, **47**, 1995, 149-171.
- [3] **J.Ostrowska-Maciejewska, J.Rychlewski**, (1988). Plane elastic and limit states in anisotropic solids, *Arch. Mech.*, **40**, 379-386.
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- [5] **P.Kordzikowski, M.Janus-Michalska, R.B.Pęcherski**, (2003). Analysis of the influence of the strength of the struts forming a cubic cell structure on the distribution of the energy limits, *Rudy i Metale Nieżelazne*, R49, No.3, 2004.